Modeling of 3D Flow and Scouring around Circular Piers

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ABSTRACT

By combining a three-dimensional (3D) flow model with a scour model, a morphological model has been constructed to simulate the flow field and bed evolution around bridge piers. The large eddy simulation (LES) approach with Smagorinsky's subgrid-scale (SGS) turbulent model is employed to compute 3D flow velocity and bed shear fields. For relatively coarse bed materials, the scour model solves the sediment continuity equation in conjunction with van Rijn's bed-load sediment transport formula to simulate the bed evolution. Without recomputing the 3D flow field as the bed deforms, the shear field obtained from the 3D flow model under flatbed conditions is modified according to the bed deformation. The 3D flow model is verified with experimental data obtained under flatbed conditions. The gravitational effect of the sloping bed of the scour hole on sediment particle movement is incorporated as part of the effective bed shear stress in the scour model. The scouring effect resulting from downflow in the region in front of the pier is included in the model by referring to the vertical jet flow scour relation. The measured data of scour evolution at the pier nose obtained by R. Ettema and bed elevation contours around a pier obtained by G. H. Lin are used for calibration and verification of the model. The results show good agreement between simulation and experimental nesults.

Key Words: pier, scour, 3D flow, downflow, bed shear stress

I. Introduction

As water flow approaches a bridge pier, it is forced to separate and pass around the pier. The flow phenomena are complex due to the presence of a boundary layer as well as an adverse pressure gradient set up by the bridge pier. Consequently, the mechanism of the local scouring processes is complicated by 3D flow patterns, such as horseshoe vortex and downward current (downflow), and bed shear distribution around the pier. Many researchers have conducted a vast number of experiments in laboratory flumes to investigate the local scour depth around a bridge pier. Quite a few empirical formulas predicting the maximum scour depth have been developed under various experimental conditions. However, most of the experiments have been carried out in flumes under idealized conditions, such as steady flow, uniform sediment, simplified geometry, etc. (Ettema, 1980; Chiew and Melville, 1987; Lin, 1993). Therefore, their applications to field situations may still be problematic and may produce questionable results. A more satisfactory approach for further applications in field situations is to simulate accurately the flow field and scouring processes using a 3D numerical model. Modeling 3D flow field and scour hole evolution around a bridge pier is more feasible nowadays because the computational cost and computational time have significantly decreased.

In recent years, several numerical models have been constructed for simulating the 3D flow field and/or bed variations around circular piers. Richardson and Panchang (1998) used a 3D transient model to compute the flow field around a pier within a given fixed scour hole. Without modeling sediment transport, they estimated the depth of equilibrium scour simply by means of Lagrangian particle-tracking analysis. By incorporating various sediment transport models, a few researchers have developed scouring models with various features. Omitting the transient terms, Olsen and Malaaen (1993) computed the scour hole development by solving the 3D Navier-Stokes equations with the κ - ε (turbulent kinetic energy and dissipation rate) model for the Reynolds stresses, and the advectiondiffusion equation for sediment transport. Olsen and Kjellesvig (1998) extended the aforementioned model of Olsen and Malaaen with transient terms. For a scouring process covering 416 hours, however, a computational time of 9 weeks on an IBM-370 workstation was required. Using a finite element method to solve the 3D Navier-Stokes equations along with a stochastic turbulence-closure model, Dou (1997) proposed a function called the sediment transport capacity for local scouring to express the effects of downflow, vortex strength and turbulent intensity in the sediment transport part. Never-

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theless, three more coefficients in the function of the sediment transport capacity for local scouring need to be determined. Roulund *et al.* (1999) simulated the scouring processes over only a very short duration (5 minutes) by using a 3D flow model and solving the sediment continuity equation with Engelund's bedload transport formula (Engelund, 1966). Tseng *et al.* (2000) investigated numerically the 3D turbulent flow field around square and circular piers. The simulated results they obtained indicated that the velocity and shear stress around the square pier were significantly higher than those around the circular pier. According to the aforementioned researches, the computational cost and time are still the major limitations for further applications when these models are used.

In the present study, a morphological model consisting of a 3D flow model and a scour model was developed to simulate the bed evolution around a circular pier. In order to reduce the computational time involved in repeatedly recomputing the 3D flow field as the bed scouring process progresses, an algorithm was developed to modify the bed shear field in order to account for bed deformation due to scouring. For the flow model, the simulated 3D flatbed flow field is compared here with experimental data obtained by Yeh (1996). In the scour model, the gravity effect of the sloping bed of the local scour hole is incorporated as part of the effective bed shear and verified by experimental results. Furthermore, in order to simulate the scouring process resulting from downflow in front of the pier, a relationship based on submerged jet flow scouring (Clarke, 1962) has been modified and employed. The experimental data for the scour depth at the pier nose obtained by Ettema (1980) and scour depth contours obtained by Lin (1993) are compared with simulated results obtained in this study to check the validity of our model.

II. Three-Dimensional Flow Model

1. Velocity Field

In order to describe the complex 3D flow patterns, including downflow in front of the pier and a horseshoe vortex around the circular pier, the weakly compressible flow theory (Song and Yuan, 1988) was employed. The large eddy simulation (LES) approach incorporated with Smagorinsky's subgrid-scale (SGS) turbulence model was adopted to simulate the flow and bed shear fields (Song and Yuan, 1990). The LES approach has gained wider acceptance for solving hydraulic problems because the SGS turbulence model is less dependent on the model coefficient than the κ - ε turbulence model (Thomas and Williams, 1995). The mathematical expressions for the weakly compressible flow equations, LES approach, SGS turbulence model, boundary conditions and numerical approach in explicit finite volume method based on MacCormack's predictor-corrector scheme are given in the Appendix.

2. Bed Shear

Generally speaking, the bed shear stress (τ_{ij}) can be calculated using the following equation (Nezu and Rodi, 1986):

$$\tau_{ij} = \mu(\frac{\partial \overline{u}_i}{\partial x_j} + \frac{\partial \overline{u}_j}{\partial x_i}) - \rho \overline{u'_i u'_j}, \qquad (1)$$

where μ is the dynamic viscosity; \overline{u}_i is the time-averaged velocity component; and $-\rho u'_i u'_i$ is the Reynolds' stress.

For a hydraulically smooth bed, the Reynolds' stress term in the viscous sublayer is much smaller than the viscous shear stress term. Hence, the Reynolds' stress term is negligible, and the bed shear stress can be calculated directly as follows:

$$\tau_{ij} = \mu(\frac{\partial \overline{u}_i}{\partial x_j} + \frac{\partial \overline{u}_j}{\partial x_i})$$
(2)

In order to calculate the bed shear stress using Eq. (2), the size of the grid mesh adjacent to the bed must be kept smaller than the thickness of viscous sublayer.

To make possible the bed shear modification made to take into account bed deformation during scouring, Taylor series expansion is applied to the logarithmic velocity profile for bed deformation of ΔZ . This leads to (Yen *et al.*, 1997)

$$\frac{\hat{U}}{\hat{u}_*} = \frac{U}{u_*} + D_z, \qquad (3)$$

in which \hat{U} is the modified depth-averaged velocity after bed deformation; \hat{u}_* is the modified shear velocity after bed deformation; U is the depth-averaged velocity before bed deformation; u_* is the shear velocity before bed deformation; $D_z = 2.5[\frac{\Delta Z}{R} - \frac{1}{2}(\frac{\Delta Z}{R})^2]$; and R is the water depth before bed deformation. Invoking the definitions $\tau = \rho u_*^2$ and $\hat{\tau} = \rho \hat{u}_*^2$, Eq. (3) becomes

$$\hat{\tau} = \tau (\frac{\hat{u}_*}{u_*})^2 = \tau \times \left[\frac{U}{\hat{U}} (1 + \frac{u_*}{U}D_z)\right]^{-2},$$
(4)

in which $\hat{\tau}$ is the modified bed shear stress after bed deformation; and τ is the bed shear stress before bed deformation.

Strictly speaking, the velocity profile in the region close to the pier no longer satisfies the logarithmic distribution. Therefore, Eq. (4) can only be applied in the region some distance away from the pier. However, the ratio of the modified bed shear to the original bed shear in the vicinity of the pier is assumed to be the same as that in the region with the logarithmic velocity profile.

III. Scour Model

1. Bed Evolution

The evolution of the scour hole can be simulated by

solving the sediment continuity equation with a sediment transport relation. Assuming that scouring takes place in the form of bedload transport, one can write the 2D sediment continuity equation as

$$\frac{\partial q_{sx}}{\partial x} + \frac{\partial q_{sy}}{\partial y} + (1 - \lambda_n) \frac{\partial Z_b}{\partial t} = 0 , \qquad (5)$$

where q_{sx} and q_{sy} are the sediment transport rates in the xand y-directions, respectively; λ_n is the sediment porosity; and Z_b is the bed elevation.

For coarse bed materials, sediment basically moves by rolling, sliding or jumping along the bed. The widely used bed-load sediment transport formula proposed by van Rijn (1986) is employed in the present study. van Rijn's bed-load transport formula is expressed as

$$q_s = 0.053 \sqrt{S'_{sg}} d^{1.5} \frac{T_*^{2.1}}{D_*^{0.3}}, \qquad (6)$$

where q_s is the sediment volume transport rate per unit width; $S'_s = (S_s - 1)$; S_s is the specific gravity of the sediment; g is the gravitational acceleration; d is the sediment diameter, T_* $= (\tau_b - \tau_c)/\tau_c$, which is called the transport stage parameter; τ_b is the bed shear stress; τ_c is the critical shear stress; $D_* = d(\rho^2 S'_s g/\mu^2)^{1/3}$, which is called the particle parameter; and μ is the dynamic viscosity.

To solve Eq. (5), open and solid boundary conditions are imposed. The upstream inflow boundary condition is given by $q_{sx} = q_{sy} = 0$ for clear water scour. The downstream outflow boundary condition is also given by $q_{sx} = q_{sy} = 0$ because at some distance downstream, the flow becomes uniform again. For solid and lateral boundaries, no sediment flux conditions ($q_{sn} = 0$, where q_{sn} is the transport rate in the direction normal to the boundaries) are imposed.

2. Effect of Local Bed Slope

In order to apply the sediment transport formula appropriately in the scour hole with a sloping bed, the gravitational component along the bed surface is considered here as a part of the effective shear stress driving the motion of the sediment particles. On the sloping bed of the scour hole, the direction of sediment motion may not coincide with the direction of bed shear due to the flow motion; it is determined by the immersed weight of the sediment particle and the bed shear on the particle. In the direction of sediment motion, therefore, the effective shear stress empolyed in van Rijn's bed-load transport formula is expressed as

$$\tau_{be} = \tau_b \times \cos(\beta - \delta) + w' \times \sin\theta \times \cos(\alpha_d - \delta)/A, \quad (7)$$

where τ_{be} is the effective shear stress; τ_b is the bed shear stress due to the flow motion; β is the angle between the direction of bed shear and the x-axis; δ is the angle between the direction of sediment motion and the x-axis, and can be evaluated using a method given elsewhere (Yen *et al.*, 1997); *w*' is the immersed weight of the sediment particle; θ is the angle of the local bed slope; α_d is the angle between the direction along the local sloping bed and the x-axis; and *A* is the projected area of the sediment particle.

In Eq. (7), the first term on the right hand side represents the effective bed shear due to flow along the direction of sediment motion, and the second term represents the effective immersed sediment weight component, again along the direction of sediment motion.

Considering a sediment particle on the sloping bed in the flow, the friction force F_f opposite to the direction of incipient sediment motion is proportional to the normal force N. Since the friction force per unit area of incipient motion is equal to the critical effective shear stress τ_c , one can write

$$\tau_c = F_f / A = k_f N / A, \tag{8}$$

where k_f is the friction coefficient, which is equal to $\tan \phi_w$, and ϕ_w is the repose angle of sediment particles in still water.

In Eq. (8), the normal force *N* acting on a sediment particle includes the immersed sediment weight component w' $\cos\theta$ and the lift force F_L caused by the flow. Therefore, Eq. (8) becomes

$$\tau_{c} = \tan \phi_{w} (\frac{w'}{A} \cos \theta - \frac{F_{L}}{A})$$
$$= \tan \phi_{w} \frac{w'}{A} \cos \theta (1 - \frac{F_{L}}{w' \cos \theta})$$
$$= \tan \phi_{w} \frac{w'}{A} \cos \theta [1 - m(\theta)], \qquad (9)$$

in which $m(\theta)$ represents the effect of the lift force of the flow, which reduces the normal force acting on a sediment particle and is obviously dependent on the local bed slope angle θ . When θ becomes large, the mean flow velocity becomes smaller due to the effects of increasing water depth and flow separation; consequently, the lift force decreases. Therefore, the coefficient $m(\theta)$ becomes smaller as θ increases. Another special case which needs to be considered is $m(\theta) = 0$, and the remaining Eq. (9), $\tan \phi_w \frac{w'}{A} \cos \theta$, simply represents the critical shear stress for sediment particles on the sloping bed in still water. The coefficient $m(\theta)$ can be calibrated in the model.

3. Effect of Downflow

As water flow approaches a pier, it is forced to form the downflow that essentially dominates the scouring process in the area immediately upstream of the pier. In order to describe the effect of the downflow on the scouring process, the submerged jet flow scouring process is adopted to model the evolution of the scour depth in the area in front of the pier.

Clarke (1962) studied the scour depth evolution generated by submerged vertical jet flow and proposed the following relations:

$$y_{sd} = (0.21 \pm 0.003)D_{c}$$

$$\frac{D_{c}}{D_{u}} = 5.5(\frac{w_{0}}{\sqrt{gD_{u}}})^{0.43} \cdot (\frac{w_{0}}{\omega})^{0.05} \cdot (\frac{gt}{\omega})^{0.05} \qquad (10)$$

where y_{sd} is the scour depth generated by submerged vertical jet flow; D_c is the diameter of the scour hole; D_u is the diameter of the jet flow; w_o is the exit velocity of jet flow; t is time; and ω is the sediment particle fall velocity.

In the present study, the downflow is considered analogous to the submerged vertical jet flow. The jet flow exit velocity w_o is replaced with the maximum downflow velocity (downflow strength) w_m , and the scour depth y_{sd} generated by the jet flow is replaced with the downflow scour depth d_j . Since the characteristics of the downflow strength which develope near the bed surface are somewhat different from those of the jet flow, Eq. (10) is modified by introducing a coefficient C_1 . Thus, one has

$$\frac{d_j}{b} = \alpha \cdot \left(\frac{w_m}{u_o}\right)^{0.48} \cdot \left(\frac{t \cdot u_o}{b}\right)^{\gamma},\tag{11}$$

where *b* is the pier diameter; u_o is the mean velocity of the approaching flow; γ is an exponent depending on w_m ; and

$$\alpha = \frac{1.155 \cdot C_1 \cdot D_u^{0.785} u_o^{0.48 - \gamma}}{g^{0.215 - \gamma} \omega^{0.05 + \gamma b^{1 - \gamma}}}.$$
 (12)

In the present study, α is a coefficient that needs to be calibrated in the scour model. Rouse's experimental results (Rouse, 1949) are used to establish a relationship between γ and w_m as follows:

$$\gamma = 0.03 \frac{w_m}{\omega} + 0.078 . \tag{13}$$

Furthermore, Ettema's experiment results (Ettema, 1980) are employed to develop, by regression, a relationship between the downflow strength and the scour depth at the pier nose:

$$w_m/w_{mo} = 1 - 0.33(d_s/b),$$
 (14)

where w_m is the downflow strength for a scour hole having a depth of d_s at the pier nose; w_{mo} is the downflow strength under flatbed conditions; and d_s/b is the ratio of the scour depth to the pier diameter. (Note that d_s/b is negative.)

In the present study, Eq. (11) is incorporated into the scour model. The increase in scour depth due to downflow, Δd_i , in one time step can be calculated using Eq. (11) first,

and then the bed deformation due to the effective shear stress, ΔZ_b , in the same time step can be computed using Eq. (5) with the bed-load transport formula. The final bed elevation at the pier nose, d_s , is the sum of Δd_i and ΔZ_b for all time steps.

4. Numerical Treatment

For numerical computation in the scour model, Eq. (5) is transformed into a conservative form as follows:

$$\frac{\partial Z_b}{\partial t} + \nabla \times H_s = 0, \qquad (15)$$

where $H_s = \frac{1}{1 - \lambda_n} [q_{sx}, q_{sy}].$

By integrating over a finite control area and invoking the divergence theorem, Eq. (15) becomes

$$\frac{\partial \tilde{Z}_b}{\partial t} = -\frac{1}{A_s} \int_{\Gamma} H_s \times \underline{n} \, d\Gamma \,, \tag{16}$$

where \tilde{Z}_b represents the averaged elevation within the finite control area; A_s is the area of the finite grid mesh; <u>n</u> is unit vector normal to the line of grid mesh; and Γ is the perimeter of the finite control area.

By using the forward difference in time, the bed elevation $(\tilde{Z}_b)_{m+1}$ in the advanced time step can be calculated as follows:

$$(\tilde{Z}_b)_{m+1} = (\tilde{Z}_b)_m - \frac{\Delta t_s}{A_s} \int_{\Gamma} H_s \times \underline{n} \, d\Gamma \,, \tag{17}$$

where the subscript *m* represents computational time, and Δt_s is the time step adopted in the scour model.

IV. Verification of the Local Bed-Slope Effect

To verify the gravity effect of the local bed slope in a scour hole, as described in the scour model (see Section III), several experiments were carried out in the present study. The experiments were conducted in a box 60 cm long by 20 cm wide. The box was first filled partially with sand with a mean diameter of 1.3 mm. The initial bed slope was set at 45° and sustained by a thin plate. Then water was slowly poured into the box, and the plate was quickly but carefully removed. After removal of the plate, sediment particles began to move down the slope mainly due to its gravitational component. The final stabilized bed slope was found to be at approximately the angle of repose of sediment in water.

Four runs (Run 1 – Run 4) of experiments under the same set of conditions were conducted repeatedly. The evolution of the bed slope was recorded by a SONY TRV16 camera recorder (Sony Co., Tokyo, Japan). The recorded film was then analyzed using the Ulead VideoStudio 3.0 SE software program (Ulead Systems Inc., Taipei, Taiwan, R.O.C.). The



Fig. 1. Simulated and measured bed profiles.



Fig. 2. Bed evolution at point A.

initial and final bed profiles for all the runs are plotted in Fig.1. The evolution of the bed elevation above the toe of the initial sloping bed (see point A in Fig. 1) is plotted in Fig.2. As can be seen in Fig. 2, the bed elevation above point A increases rapidly during the early stage, and then begins to level off at about 1.5 sec.

Numerical simulation for the experiments described above was carried out under an initial bed slope of 45°. The bed profile evolution was simulated by solving Eq. (5) incorporated with Eq. (6). In the simulation, the effective shear stress was calculated using Eq. (7), and the critical shear stress was calculated using Eq. (9). It was assumed that the mean velocity caused by the removal of the plate was negligible, and that eddies generated by the plate decayed very rapidly. Under these assumptions, the value of τ_b in Eq. (7) was taken as zero. For the critical effective shear stress τ_c , the coefficient $m(\theta)$ in Eq. (9) representing the effect of eddies on the reduction of the normal force was mainly dependent on the decay time of eddies rather than on the change of θ . The initial value of $m(\theta)$ was calibrated to 0.91, which corresponded to $\tau_c =$ 0.74 N/m² obtained from the Shields diagram, and then $m(\theta)$ rapidly approached zero as eddies died out due to their rapid decay. The simulated bed profile at t = 2 sec. and the evolution of the bed elevation above point A are plotted in Figs. 1 and 2, respectively. In Fig. 2, one can see that the simulated bed evolution and equilibrium time (at 1.5 sec) agree well with the measured data. These results clearly show that treating the gravitational component along the sloping bed of the local scour hole as a part of the effective shear stress driving the motion of the sediment particle is quite reasonable.

V. Flow Field Simulation

For validation of the 3D flow model, the experimental results obtained by Yeh (1996) were compared with simulation results. The conditions for the experiment were as follows: the pier diameter b = 0.031 m; the approaching surface velocity $u_o = 0.126$ m/s; the approaching water depth $h_o = 0.082$ m; the bed slope $S_o = 0.0013$; and the bed shear velocity $u_* = 6.83 \times 10^{-3}$ m/s.

Numerical simulation was carried out under the conditions stated above. A structured grid mesh on the x-y plane was generated by an elliptic grid generator (Tseng, 1994). As shown in Fig. 3, a two-dimensional grid mesh with 60 elements in the x-direction and 36 elements in the y-direction was drawn, and there were 22 elements of nonuniform size in the zdirection. The smallest grid element was $0.028b \times 0.021b$ $\times 0.003b$. The simulation domain was $-2 \le x/b \le 9.5, -2 \le x/b \le 9.5$ $y/b \le 2$, and $0 \le z/b \le 2.65$. The upstream boundary condition was given by the measured velocity profile along the zdirection. The downstream boundary condition was given by the Neumann condition $(\partial u/\partial x = \partial v/\partial x = \partial w/\partial x = 0)$. The boundary condition on the sides (i.e., $y/b = \pm 2$) was given by the fully slip condition $(\partial u/\partial y = \partial w/\partial y = 0 \text{ and } v = 0)$. For the solid boundary condition, the cells adjoining the bed were given by the no-slip condition, and the cells adjoining the pier surface were prescribed by the partial slip condition in order to consider the variation of the local thickness of boundary layer relative to the local cell size (Song and Yuan, 1990). A value of 0.13 for the Smagorinsky constant was selected, which was within a reasonable range $(0.094 \sim 0.2)$ for solving open channel flow problems (Thomas and Williams, 1995).

Some of the simulated results are plotted in Figs. 4 – 6. Since the flow field at section x/b = 9.5 is almost the same



Fig. 3. Projection of the grid mesh on the x-y plane and grid sized in the z-direction.

as that of the upstream inflow, the simulated results are only plotted within the range of $-2 \le x/b \le 3$ in the x-direction to clearly show the detailed flow field around the pier. Figure 4(a) shows the simulated velocity field on the vertical plane section (x-z plane) along the centerline. A downflow along a vertical line near the pier nose and a circulation area, as a component of a horseshoe vortex, close to the bed can be clearly seen. Figure 4(b) shows the simulated velocity field on the vertical plane (y-z plane) 90° from the centerline. One can clearly see that the velocity near the bed is enhanced. This may have significant implications for the initial stage of scouring.

Figure 5(a) presents the simulated velocity field on the horizontal plane at z/b = 0.032. It clearly shows that reverse flow exists in front of the pier. A comparison between the measured and simulated isovels at z/b = 0.032 above the bed, presented in Fig. 5(b), indicates good agreement between them. From these results, one can see that the 3D flow model can simulate flow features around the pier very well.

Upon obtaining the velocity field, the bed shear stresses were computed in the 3D flow model using Eq. (2). The simulated bed shear stresses along the centerline upstream of the pier are plotted in Fig. 6 and compared with the measured





Fig. 5. (a) Simulated velocity field on the horizontal plane at z/b = 0.032. (b) Comparison of simulated (upper half) and measured (lower half) isovels on the horizontal plane at z/b = 0.032 around the pier.



Fig. 6. Comparison of simulated and measured bed shear stresses along the centerline in front of the pier.

Fig. 4. (a) Simulated velocity field on the vertical plane section (x-z plane) along the centerline. (b) Simulated velocity field on the vertical plane (y-z plane) 90° from the centerline.

data (Yeh, 1996). Another simulated result plotted as the dotted-line in Fig. 6 is from a previous work (Yen *et al.*, 1997). It is based on the assumption that the velocity profile is of

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Table 1. Conditions of Experiments Adopted for Scour Simulations

	Ettema (1980)	Lin (1993)
Approaching flow velocity u_o (m/s)	0.71	0.65
Approaching flow depth h (m)	0.60	0.39
Diameter of bridge pier b (m)	0.24	0.088
Diameter of sediment d (mm)	1.90	2.50
Ratio of approaching shear velocity to critical shear velocity u_*/u_*c	0.90	0.85



Fig. 7. Coefficient $m(\theta)$ calibrated by means of Ettema's experiments.

logarithmic distribution for computation of the bed shear, and it does not fit the measured data in the region of -1.5 < x/b < -0.5. In this region, one can find that the bed shear stress varies from positive to negative at the point of flow separation, which is located approximately at x/b = -1.25. The location of this separation point, however, falls slightly upstream from the range of $-1.1 \le x/b \le -0.7$ given by Graf and Yulistiyanto (1998). Nevertheless, the bed condition for the present simulation is hydraulically smooth (Yeh, 1996); hence, the turbulence intensity in the flow may not be strong enough to transfer sufficient momentum into the vicinity of the bed surface to push the separation point further downstream.

VI. Scour Simulation

For calibration and verification of the scour model, the experimental conditions used and results obtained by Ettema (1980) and by Lin (1993) were employed for simulation and comparison. The flow and sediment conditions used in their experiments are listed in Table 1.

Ettema's experiments were used to calibrate the model coefficients. After a 15-hour $(tu_o/b = 160,000)$ scour simulation run, the coefficient α in Eq. (11) was calibrated and found to be 0.5, and the coefficient $m(\theta)$ in Eq. (9) was calibrated as shown in Fig. 7. In Fig. 8, the simulated scour depth at the pier nose as a function of time is compared with the measured data, and one can see rather good agreement

between them.

For verification of the scour model, the calibrated coefficients were used to simulate the scour depth evolution in Lin's experiments. The simulated scour depth at the pier nose is plotted in Fig. 9, and the measured scour depth at 2 hours $(tu_0/b = 53,000)$ is also plotted in the figure to show the good match. The ratio of the maximum scour depth to the water depth is about 1/6 in this case, and this parameter is mainly affected by the ratio of the approaching velocity to the critical velocity, the ratio of the particle diameter to the pier diameter, the ratio of the approaching water depth to the pier diameter, etc. Furthermore, the simulated and measured final bed elevation contours are shown in Fig. 10 for comparison. The scour hole extends around the circular pier with a deeper area at the upstream side and a shallower one downstream. With the exception of a small region on the downstream side, the simulated results of the scour pattern and maximum scour depth are satisfactory, and the overall simulation is fairly good. In the wake region close to the pier where turbulence is relatively strong, the effective critical shear stress for sediment motion may be significantly reduced. This fact has not been accounted for in the simulation model; therefore, the simu-



Fig. 8. Comparison of simulated and experimental results of the scour depth evolution at the pier nose – a calibration run.



Fig. 9. Comparison of simulated and experimental results of the scour depth evolution at the pier nose – a verification run.



Fig. 10. Comparison of measured and simulated final bed elevation contours – a verification run.

lation yields slightly less scouring in the small region behind the pier.

VII. Summary and Conclusions

The morphological model consisting of a 3D flow model with a scour model has been constructed to simulate the bed scour evolution around a bridge pier. For 3D flow modeling, the LES approach with Smagorinsky's SGS turbulence model has been employed to compute the velocity and bed shear stress fields. In the scour model, the sediment continuity equation incorporated with van Rijn's bed-load sediment transport formula for relatively coarse materials has been solved to obtain the bed evolution.

The 3D flow model has been validated using experimental data measured using a Fiber-Optic Laser Doppler Velocimeter under flatbed conditions (Yeh, 1996), and a Smagorinsky constant of 0.13 was selected, which falls reasonably well within the range (0.094 \sim 0.2) in open channel flow problems (Thomas and Williams, 1995). From the simulation results, it has been found that the 3D flow model can simulate the velocity field around the pier very well.

The effect of the local bed slope of the scour hole on the direction of sediment particle motion has been incorporated into the model as part of the effective shear stress. Experiments as well as simulation were conducted to verify this effect. The effect of downflow on the scour depth has also been included in the model by referring to the submerged vertical jet flow scour relation in order to improve the simulation in the area in front of the pier. The experimental data of the scour depth evolution at the pier nose obtained by Ettema (1980) and the scour depth contours obtained by Lin (1993) have been compared with the simulated results and show good agreement. On the basis of the results presented in the present study, the morphological model proposed herein is regarded as feasible for further applications in field situations.

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APPENDIX Basic Theory for 3D Flow Simulation

1. Governing Equations

According to weakly compressible flow theory (Song and Yuan, 1988, 1990), the Mach number in water flow is so small that the fluid density and sound speed may be regarded as constants without causing significant error. It is noted that weakly compressible flow is equivalent to hydraulic transient flow. Therefore, the equation of state for a fluid may be represented by the following linear relation:

$$p - p_o = a_o^2 (\rho - \rho_o),\tag{A1}$$

where p is the pressure; a is the speed of sound; ρ is the density of the fluid; and the subscript "o" represents the reference value.

Substituting Eq. (A1) into the equations of continuity and motion for compressible fluids, the equations become

$$\frac{\partial p}{\partial t} + \boldsymbol{u}_j \frac{\partial p}{\partial x_j} + \rho_o a_o^2 \frac{\partial \boldsymbol{u}_j}{\partial x_j} = 0$$
 (A2)

$$\frac{\partial(\alpha \boldsymbol{u}_i)}{\partial t} + \frac{\partial(\alpha \boldsymbol{u}_i \boldsymbol{u}_j)}{\partial x_j} + \frac{1}{\rho_o} \frac{\partial p}{\partial x_i} = \upsilon \frac{\partial^2 \boldsymbol{u}_i}{\partial x_j \partial x_j} + \frac{\upsilon}{3} \frac{\partial}{\partial x_i} (1 - \alpha), \quad (A3)$$

where *u* is the velocity vector; x_i and x_j are coordinates; v is the kinematic viscosity of the fluid; *t* is time; $\alpha = 1 + \frac{p - p_o}{\rho_o a_o^2} = 1 + C_p M^2$; C_p is the pressure coefficient; and *M* is the Mach number.

When $M \ll 1$, the pressure term in Eq. (A2) is negligible and $\alpha \approx 1$. Then Eqs. (A2) and (A3) can be rewritten as

$$\frac{\partial p}{\partial t} + \rho_o a_o^2 \frac{\partial u_j}{\partial x_j} = 0$$
(A4)

$$\frac{\partial \boldsymbol{u}_i}{\partial t} + \frac{\partial (\boldsymbol{u}_i \boldsymbol{u}_j)}{\partial x_j} + \frac{1}{\rho_o} \frac{\partial p}{\partial x_i} = v \frac{\partial^2 \boldsymbol{u}_i}{\partial x_j \partial x_j} .$$
(A5)

Equations (A4) and (A5) are defined as weakly compressible flow equations, which were also used for hydraulic transient flow as the primary governing equations in the present study.

Generally speaking, Eq. (A5) also can be transformed into vorticity equation form and then can be solved to obtain the flow field. However, solving the vorticity equation to simulate the flow field around piers may be more difficult because the initial and boundary conditions are not easy to prescribe.

2. Large Eddy Simulation

For a high Reynolds number, the turbulent flow contains a wide range of eddy sizes. Herein, the concept of large eddy simulation (LES) is adopted so as to directly calculate the large-scale turbulence that can be resolved within the computational mesh size and to model only the unresolvable smallscale turbulence.

In the mesh grid, each flow variable *f* (velocity component \boldsymbol{u}_i or pressure *p*) can be represented by an averaged quantity \overline{f} plus a quantity *f* related to subgrid scale turbulence, that is, $f = \overline{f} + f^*(\overline{f} = \frac{1}{\Delta V} \int_{\Delta V} f(V, t) \times dV$,

where ΔV is the finite volume of the computational mesh grid and V is its volume). Based on the above relationship of the averaged quantity in the finite volume, the weakly compressible flow equations become (Song and Yuan, 1988)

$$\frac{\partial \hat{p}}{\partial t} + \rho_o a_o^2 \frac{\partial \overline{u_j}}{\partial x_j} = 0, \qquad (A6)$$

$$\frac{\partial \overline{u_i}}{\partial t} + \frac{\partial (\overline{u_i} \,\overline{u_j})}{\partial x_j} + \frac{1}{\rho_o} \frac{\partial \hat{p}}{\partial x_i} = v \frac{\partial^2 \overline{u_i}}{\partial x_j \partial x_j} - \frac{1}{\rho_o} \frac{\partial \hat{\tau}_{ij}}{\partial x_j}, \tag{A7}$$

where $\hat{p} = \overline{p} + \frac{1}{3}\rho \overline{u'_k u'_k}$ is the modified pressure; and $\hat{\tau}_{ij} = \rho \overline{u'_i u'_j} - \frac{1}{3}\rho \delta_{ij} \overline{u'_k u'_k}$ is the subgrid scale turbulence stress. The subgrid turbulence stress is modeled by the following relation (Smagorinsky, 1963):

$$_{ij} = -2\rho v_t \overline{S}_{ij} , \qquad (A8)$$

where $v_t = (C\Delta_s)^2 (2\overline{S_{ij}} \overline{S_{ij}})^{1/2}$ is the subgrid eddy viscosity coefficient; $\overline{S_{ij}} = \frac{1}{2} (\partial \overline{u_i} \partial x_j + \partial \overline{u_j} \partial x_i)$ is the resolvable rate of deformation tensor; Δ_s is the subgrid length scale dependent on the mesh size; and *C* is the Smagorinsky constant, which is the only adjustable parameter in the subgrid scale turbulence (SGS) model.

3. Boundary Conditions

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To solve Eqs. (A6) and (A7), adequate boundary conditions should be imposed. Typical boundary conditions are described as follows: (1) **Upstream boundary condition**:

$$u = u_o(z); \ v = w = \frac{\partial \hat{p}}{\partial x} = 0 ;$$
 (A9)

(2) Downstream boundary condition:

$$\frac{u}{x} = \frac{\partial v}{\partial x} = \frac{\partial w}{\partial x} = 0; \quad \hat{p}_d = \hat{p}_o \quad \text{or} \quad \frac{\partial \hat{p}}{\partial x} = 0, \quad (A10)$$

where u_o is the approaching velocity; \hat{p}_d is the averaged pressure downstream; and \hat{p}_o is the reference pressure. In the present study, the downstream boundary condition is given by $\frac{\partial u}{\partial x} = \frac{\partial v}{\partial x} = \frac{\partial w}{\partial x} = 0$; $\hat{p}_d = \hat{p}_o$.

- (3) Solid boundary condition: The fully slip condition, partial slip condition (wall function) or non-slip condition can be adopted according to the characteristics of the solid boundary (Tseng, 1994; Yen *et al.*, 1997).
- (4) Free surface boundary condition: The kinematic boundary condition or dynamic boundary condition is imposed.
 - For the kinematic boundary condition, the deformation rate of the water surface must equal the flow velocity normal to the water surface, i.e.,

$$\frac{\partial Z_f}{\partial t} + \underline{u} \times \nabla Z_f = \underline{u} \times \underline{n} \quad \text{on } z = Z_f(x, y, t), \tag{A11}$$

where Z_f is the elevation of the water surface; \underline{u} is velocity vector; and \underline{n} is the unit vector normal to the water surface.

(ii) For the dynamic boundary condition, there is no momentum flux through the water surface, i.e.,

$$\boldsymbol{\sigma}_{ij}\boldsymbol{n}_{ij} = 0 \text{ on } \boldsymbol{z} = \boldsymbol{Z}_f(\boldsymbol{x}, \, \boldsymbol{y}, \, \boldsymbol{t}), \tag{A12}$$

where σ_{ij} is the stress tensor defined as $\sigma_{ij} = -p \delta_{ij} + \tau_{ij}$.

4. Numerical Approach

The finite volume method adopted in the model is the natural method

most compatible with the volume-averaged LES approach and has sufficient flexibility to fit irregular boundaries. In the framework of the finite volume method, MacCormack's explicit predictor-corrector scheme (MacCormack, 1969), which is of second order accuracy in both time and space, is employed to solve the governing equations.

Equations (A6) and (A7) can be rewritten in a conservative form:

$$\frac{\partial U}{\partial t} + \frac{\partial E}{\partial x} + \frac{\partial F}{\partial y} + \frac{\partial G}{\partial z} = 0, \text{ or } \frac{\partial U}{\partial t} + \nabla \cdot H = 0,$$
(A13)

where

$$\begin{split} U &= \left[\hat{\rho}, \overline{u}, \overline{v}, \overline{w}\right]^{T}, H = [E, F, G], \\ E &= \left[\rho_{o}a_{o}^{2}\overline{u}, \overline{u}^{2} + \frac{\hat{p}}{\rho_{o}} - v\frac{\partial\overline{u}}{\partial x} + \frac{\hat{\tau}_{xx}}{\rho_{o}}, \overline{uv} - v\frac{\partial\overline{v}}{\partial x} + \frac{\hat{\tau}_{yx}}{\rho_{o}}, \\ \overline{uw} - v\frac{\partial\overline{w}}{\partial x} + \frac{\hat{\tau}_{zx}}{\rho_{o}}\right]^{T}, \\ F &= \left[\rho_{o}a_{o}^{2}\overline{v}, \overline{uv} - v\frac{\partial\overline{u}}{\partial y} + \frac{\hat{\tau}_{xy}}{\rho_{o}}, \overline{v}^{2} + \frac{\hat{p}}{\rho_{o}} - v\frac{\partial\overline{v}}{\partial y} + \frac{\hat{\tau}_{yy}}{\rho_{o}}, \\ \overline{vw} - v\frac{\partial\overline{w}}{\partial y} + \frac{\hat{\tau}_{zy}}{\rho_{o}}\right]^{T}, \\ G &= \left[\rho_{o}a_{o}^{2}\overline{w}, \overline{uw} - v\frac{\partial\overline{u}}{\partial z} + \frac{\hat{\tau}_{xz}}{\rho_{o}}, \overline{wv} - v\frac{\partial\overline{v}}{\partial z} + \frac{\hat{\tau}_{yz}}{\rho_{o}}, \\ \overline{w}^{2} + \frac{\hat{p}}{\rho_{o}} - v\frac{\partial\overline{w}}{\partial z} + \frac{\hat{\tau}_{zz}}{\rho_{o}}\right]^{T}. \end{split}$$

After integrating over a finite control volume ΔV and using the divergence theorem, Eq. (A13) can be replaced by

$$\frac{\partial \tilde{U}}{\partial t} = I , \qquad (A14)$$

where $I = -\frac{1}{\Delta V} \int_{s} H \times \underline{n} \, ds$; \tilde{U} represents a mean quantity within the finite control volume (the grid mesh size); and *s* is its finite control surface.

With MacCormack's predictor-corrector scheme, Eq. (A14) can be

approximated in two calculation steps as follows: (1) the predictor step:

 $\tilde{U}_{m+1} = \tilde{U}_m + I_m \Delta t \; ; \qquad$

$$V_{m+1} = U_m + I_m \Delta t ; \tag{A15}$$

(2) the corrector step:

$$\tilde{U}_{m+1} = \tilde{U}_m + \frac{1}{2}(I_m + \hat{I}_{m+1})\Delta t$$
, (A16)

where $\hat{I}_{m+1} = -\frac{1}{\Delta V} \int_{s} \hat{H}_{m+1} \times \underline{n} \, ds$; the subscript *m* represents the computation time; and the hat ^ represents the predicted values.

For numerical stability, the time step Δt in Eqs. (A15) and (A16)

must be determined by using the Courant-Friedrich-Lewy condition (Song and Yuan, 1988):

$$\Delta t \le C_r \cdot \operatorname{Min}\left[\frac{\Delta V}{\left|u_i\Omega_i\right| + a_o\left|\Omega\right|}\right],\tag{A17}$$

where C_r is the Courant stability factor with a value of 0.8 adopted in the present study; u_i is the velocity vector; $|\Omega|$ is the surface area of the finite volume; and Ω_i is the projected area of $|\Omega|$ in the *i* direction.

橋墩周圍三維流場及沖刷之模擬

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摘要

本文將三維流場模式與沖刷模式結合成河床形態模式,用來模擬橋墩周圍流場及其沖淤所造成的床形變化。三維流場模式採用大渦模擬(large eddy simulation)方法,配合斯馬格林斯基(Smagorinsky)之次網格紊流模式,模擬橋墩周圍之三維流速與底床剪應力場。針對較大粒徑之均匀沈滓,沖刷模式採用范瑞因(van Rijn)之沈滓輸送公式結合沈滓連續方程式,求解橋墩周圍之床形變化。在不同沖刷階段下,底床剪應力場是由平床剪應力場修正而得,並不需重新計算該床形下之三維流場。本研究採用平床流場之試驗資料來驗證三維流場模式。在沖刷模式中,局部床形之重力效應納入為有效剪應力之一部分;而墩前下向流之沖刷效應是根據射流沖刷關係進行處理。最後,本文分別採用葉特馬(R. Ettema)與林景輝(G.H. Lin)之試驗資料來檢定與驗證沖刷模式。